

INTRODUCTION TO ENERGY HARVESTING AND LINEAR VIBRATION ENERGY HARVESTING (ICT-ENERGY SUMMER SCHOOL 2016)

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Outline for Short Course

- **Introduction and Linear Energy Harvesting**
- Energy Harvesting Transducers
 - Electromagnetic
 - Piezoelectric
 - Electrostatic
- Wideband and Nonlinear Energy Harvesting
- Applications

Outline for This Lecture

- What is energy harvesting? And, why?
- Types of energy harvesting
 - RF
 - Photovoltaic
 - Thermal
 - Wind
 - Vibrations
- Linear Vibration Energy Harvesting Theory
- Figures of Merit for Vibration Energy Harvesters

Billion Sensors



APPLE WATCH



APPLE WATCH SPORT



APPLE WATCH EDITION

- Driven by mobile devices
- Can be regularly recharged
- Short lifetime
- Not dependent on energy harvesting

Trillion Sensors

A Central Nervous System for the Earth (CeNSE)



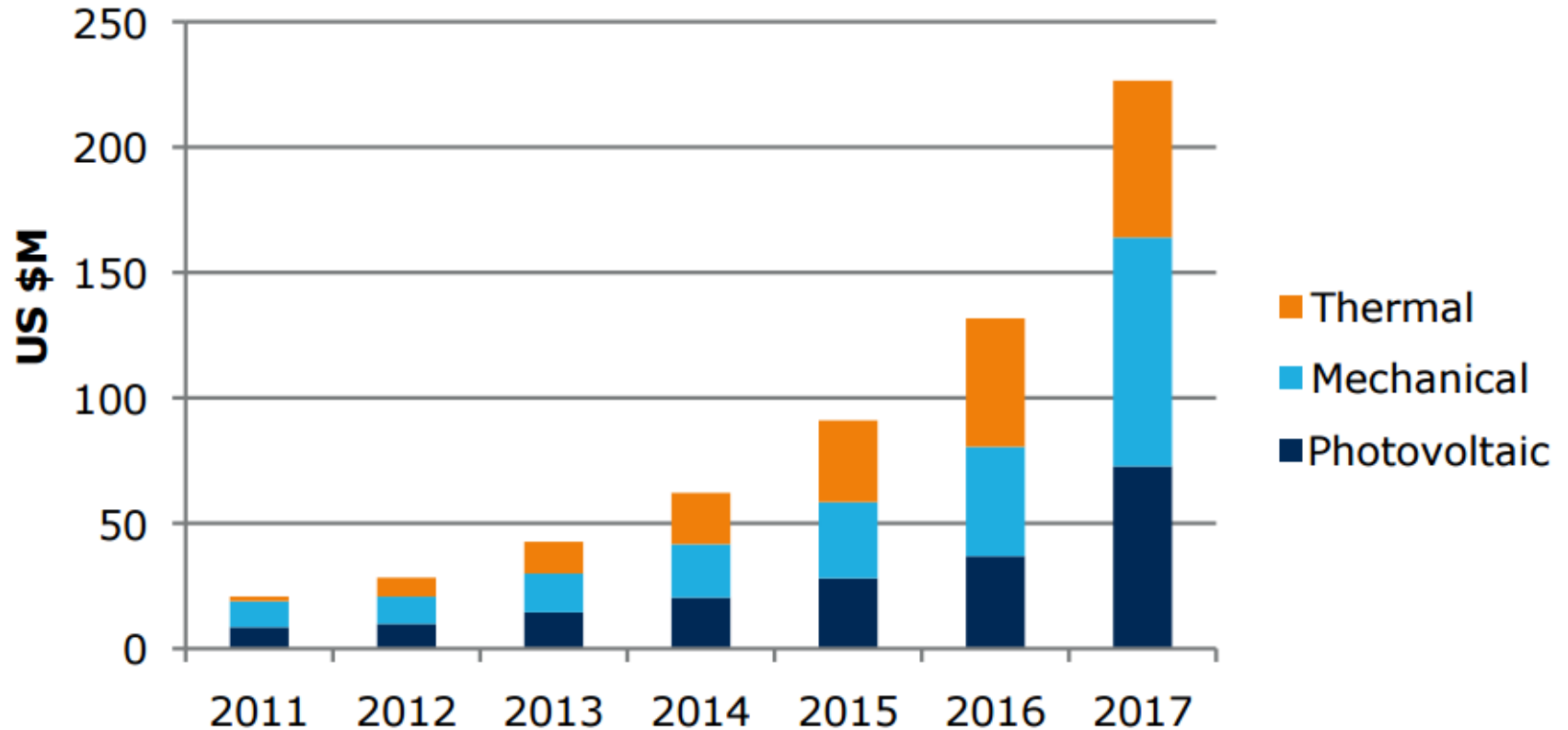
Source: Peter Hartwell, HP Labs

<http://www.hpl.hp.com/news/2009/oct-dec/cense.html>

- Driven by autonomous wireless sensors
- Long lifetime (> 5 years)
- Battery recharging costly and/or infeasible

Market for modules

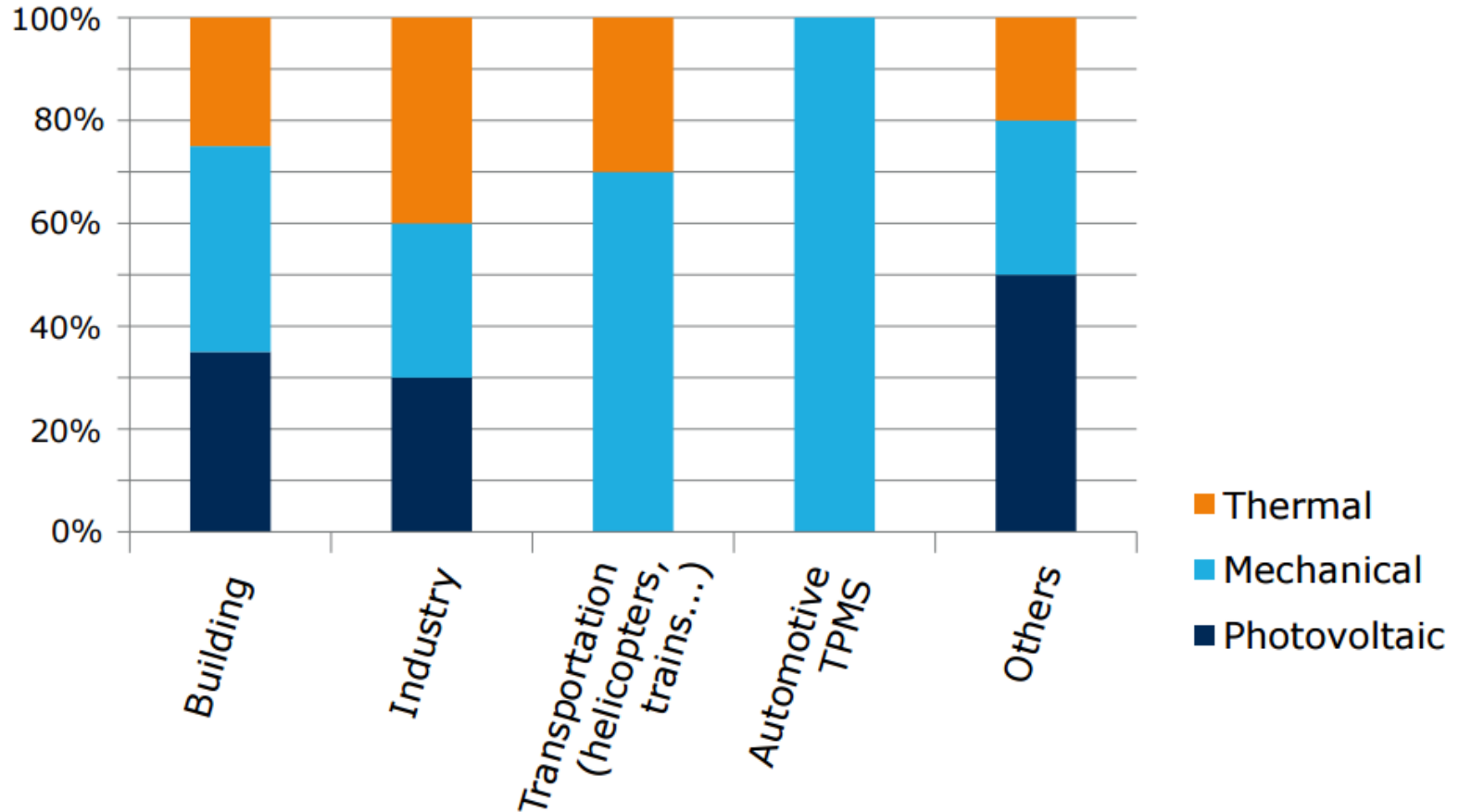
(Source: Energy Harvesting report, Yole Développement, to be released in November 2012)



http://www.yole.fr/iso_upload/Mag/ACMEMStrendsOct%202013.pdf

2017 energy harvesting transducer principle breakdown

(Source: Energy Harvesting report, Yole Développement, to be released in November 2012)



http://www.yole.fr/iso_upload/Mag/ACMEMStrendsOct%202013.pdf

IoT Device Taxonomy

(Courtesy of Vijay Raghunathan, Purdue Univ.)

Type-I

Wearable Devices



Type-II

Set-and-forget devices



Type-III

Infrastructure monitoring devices



Type-IV

Batteryless, transiently powered devices



Type-V

Wall-powered appliances



IoT Device Taxonomy

(Courtesy of Vijay Raghunathan, Purdue Univ.)

Type-I

Wearable Devices



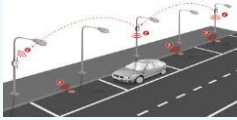


Type-II

Set-and-forget devices




Type-III

Infrastructure monitoring devices





Type-IV

Batteryless, transiently powered devices

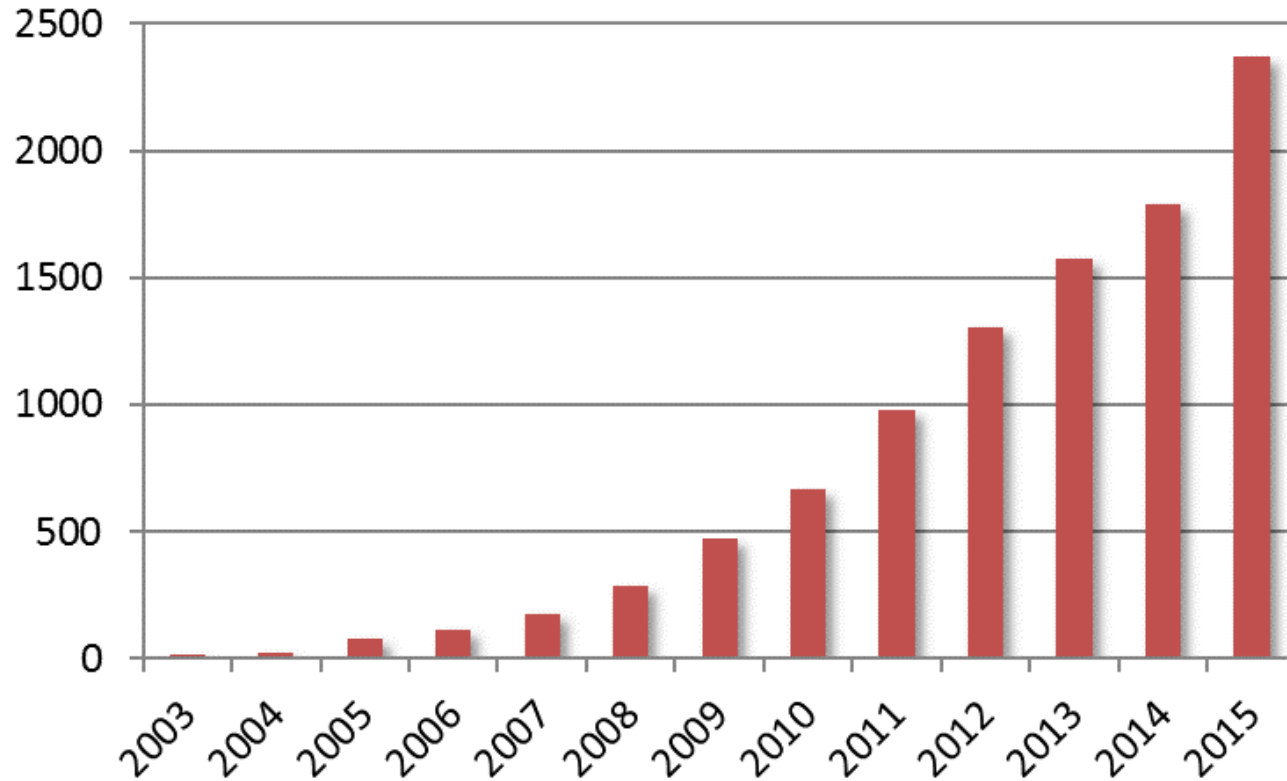


Type-V

Wall-powered appliances

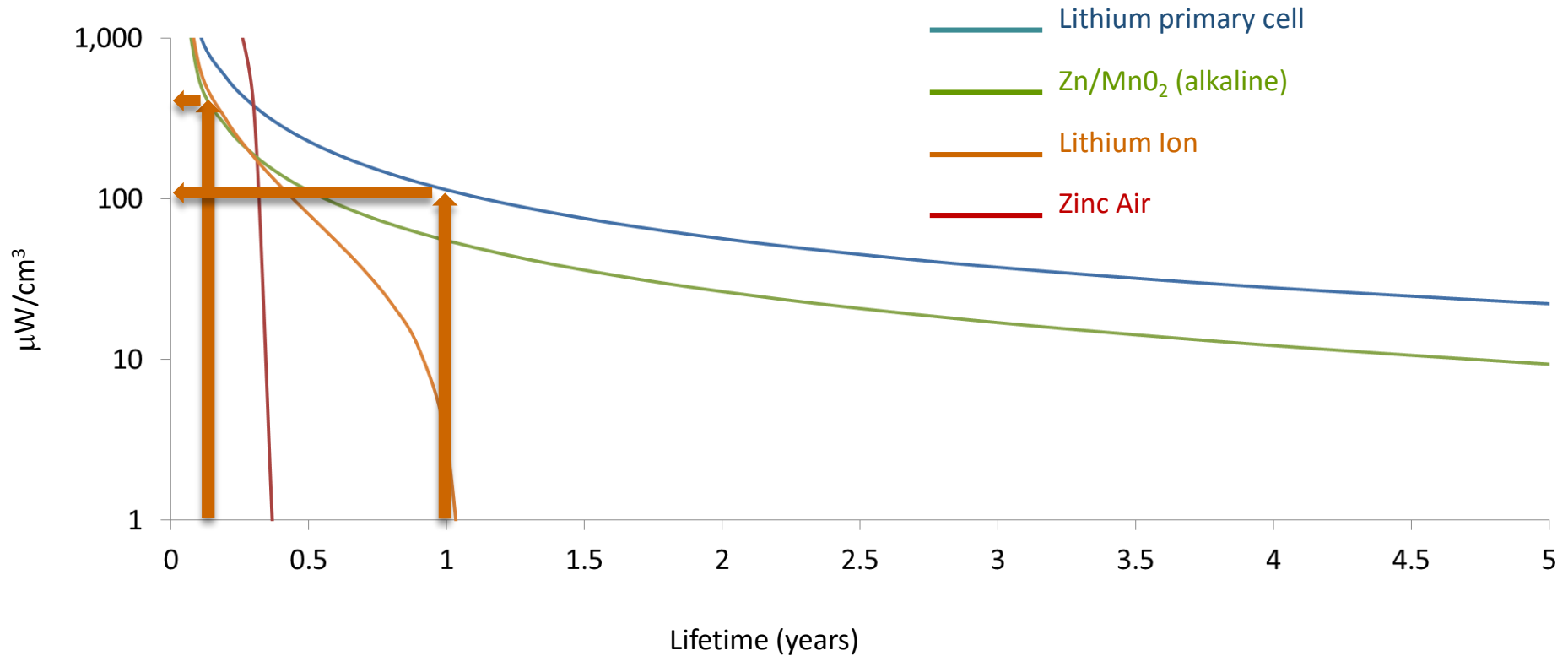


Energy Harvesting Publications by Year



From Google Scholar
“Energy Harv*” or “Energy Scav*” in title

Continuous Power / cm³ vs. Lifetime



DIFFERENT TYPES OF ENERGY HARVESTING

RF Power

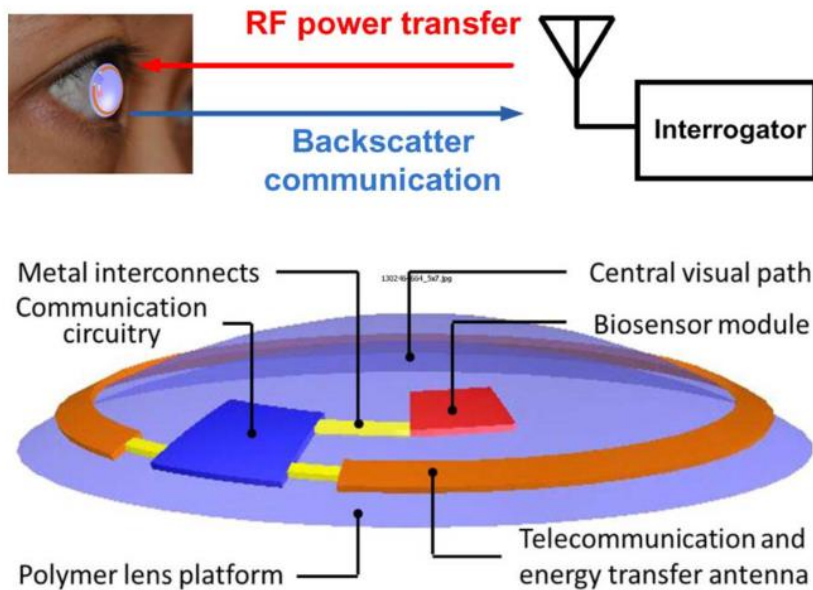


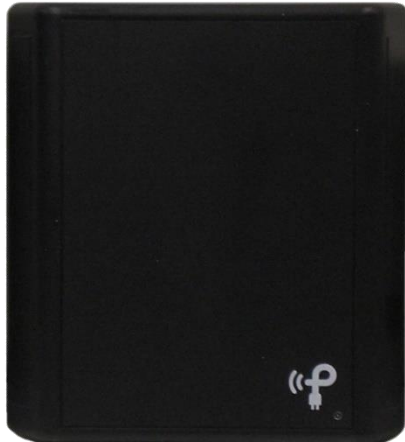
Fig. 1. Conceptual diagram of an active contact-lens system for wireless health monitoring.

Liao et. al., IEEE Journal of Solid State Circuits, 2012
Univ. of Washington

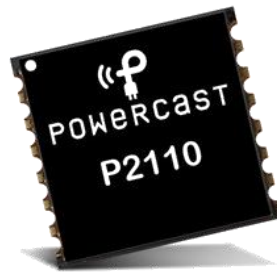
- Glucose sensor on a contact lens
- Measures glucose in tear fluid
- RF powered
- Consumes 3 μW
- Google commercializing this (https://en.wikipedia.org/wiki/Google_Contact_Lens)

RF Power

Transmitter



Receiver



P2110 – 0.53 X 0.55 inches

- Powercast makes RF power harvesting products
- Microwatts to milliwatts
- Targeting things like high functionality RFID, ePaper labels, wireless sensors etc.

<http://www.powercastco.com/power-calculator/>

RF Power

○ Example 1

- Assume antenna gains of 1

$$P_r = P_0 \left(\frac{\lambda}{4\pi R} \right)^2$$

P_0 = transmit power
 P_r = power received
 λ = wavelength
 R = transmit distance

- For 2.4 GHz, $R = 5$ meters, and $P_0 = 1$ watt

$$\bullet P_r = 4 \text{ uW}$$

• Example 2

- Use PowerCast Calculator

$$P_r = G_t G_r P_0 \left(\frac{\lambda}{4\pi R} \right)^2$$

$$f = 915 \text{ MHz}$$

$$P_0 * G_t = 1 \text{ watt}$$

$$G_r = 6.0 \text{ dBi (default value)}$$

$$R = 5 \text{ meters}$$

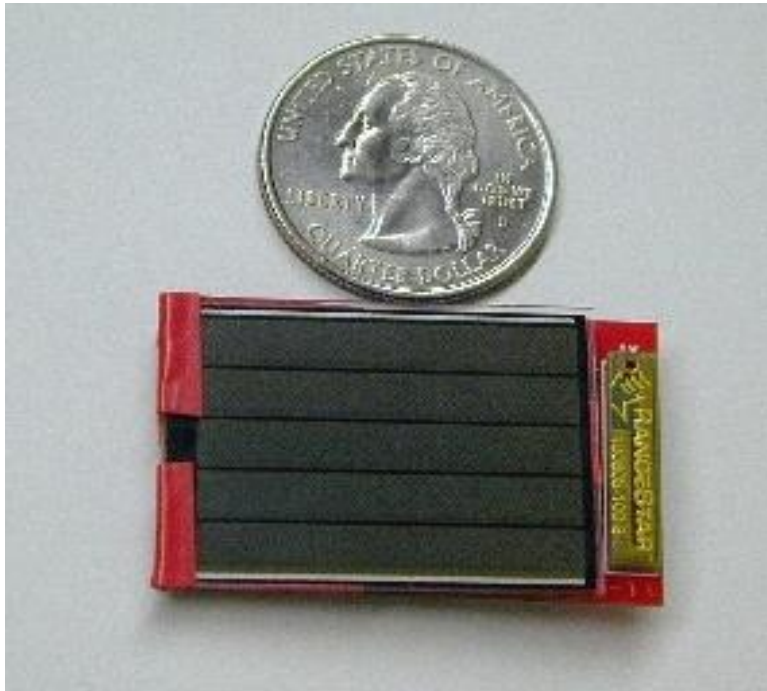
$$- P_r = 108 \text{ uW}$$

28 uw after conversion to DC

Photovoltaic

- $\sim 15 \text{ mW/cm}^2$ outside in the Sun

Inside



Power Measured At Increasing Distances from a 60 watt Bulb

Distance	Power (mW/cm ²)
20 cm	503
30 cm	236
45 cm	111
Office Light	7.2

Thermal Gradients

Even with low efficiency, power output is attractive for some applications



Citizen Eco-Drive Thermo

13.8 μW at ΔT of 1 $^{\circ}\text{C}$

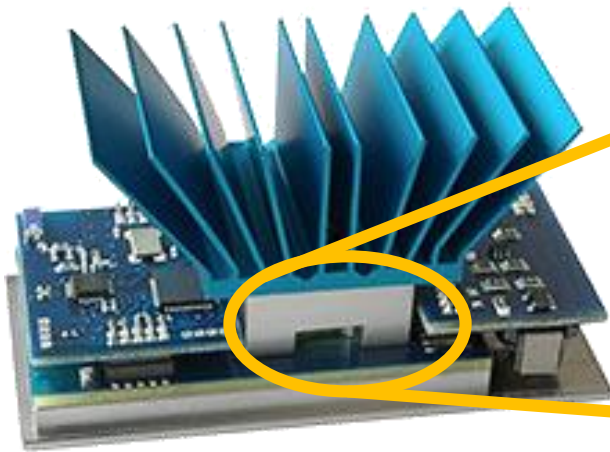
Size = 0.7cm X 0.7cm



Thermo Life

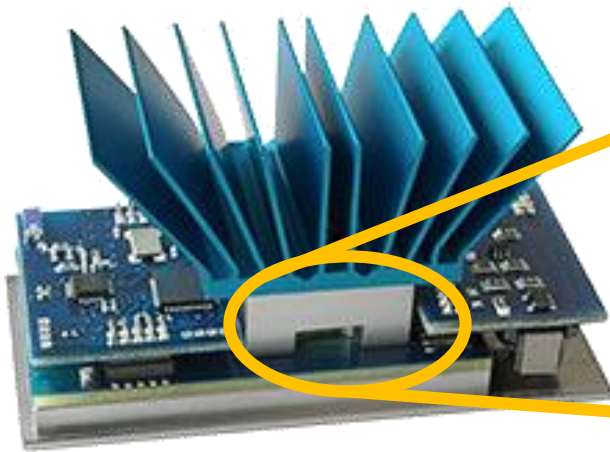
$\sim 25 \mu\text{W}$ at ΔT of 2 $^{\circ}\text{C}$

Thermal Gradients



Micropelt GmbH

Thermal Gradients



Micropelt GmbH

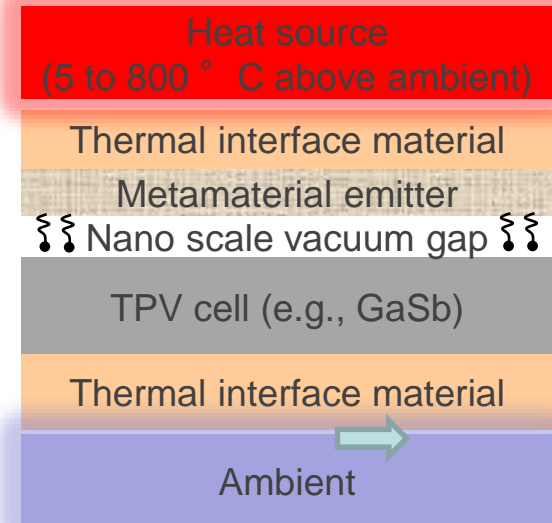
- Example using <http://www.micropelt.com/products/mypelt.php>
 - $\Delta T = 5 \text{ }^\circ\text{C}$, 25 rows in series
 - Output
 - 88 mA, 0.75 V
 - 33 mW with matched load

University of Utah nano-TPV (Courtesy of Prof. Mathieu Francoeur)

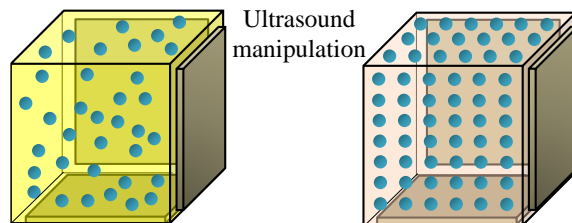
- Waste Heat Conversion

- 58% of power generated in US is lost to heat
- Potential heat recovery application spaces
 - Electronics (including data centers)
 - PV cells - passive cooling/heat recovery
 - Power generation (power plants)
 - Body heat powered electronics / sensors
 - Automotive applications

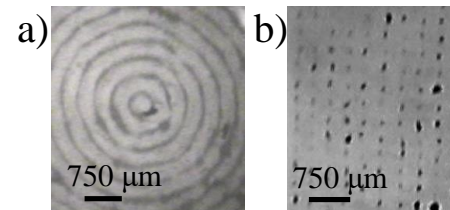
nano-TPV Stack



**Metamaterial manufacturing process:
using bulk acoustic waves for directed
self-assembly of dispersed dielectric
inclusions in a dielectric matrix**

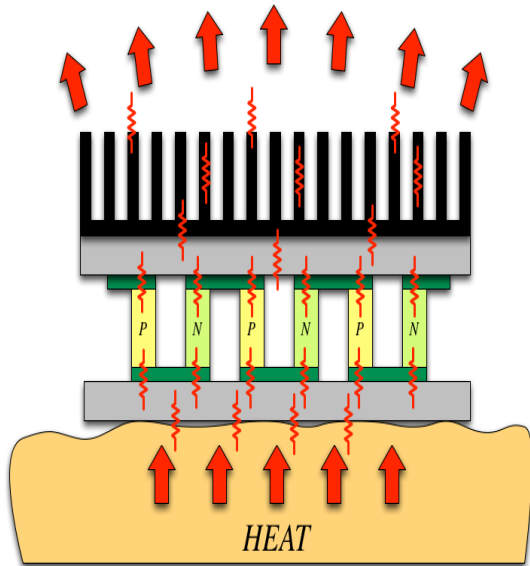


**Directed self-assembly of 5 nm diamond
particles using bulk acoustic waves**

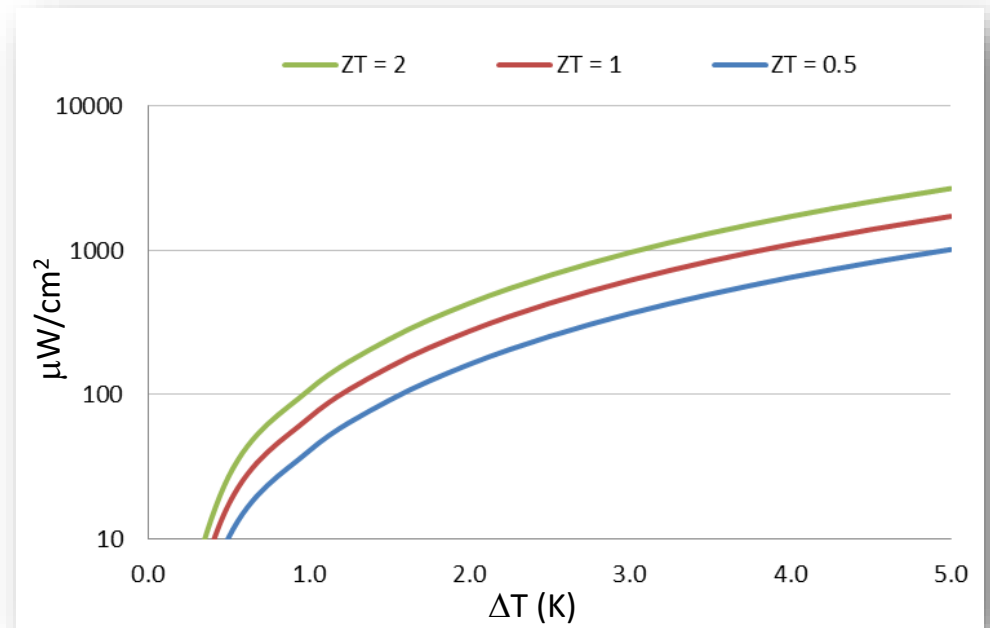


Raeymaekers et al., JAP, 2011

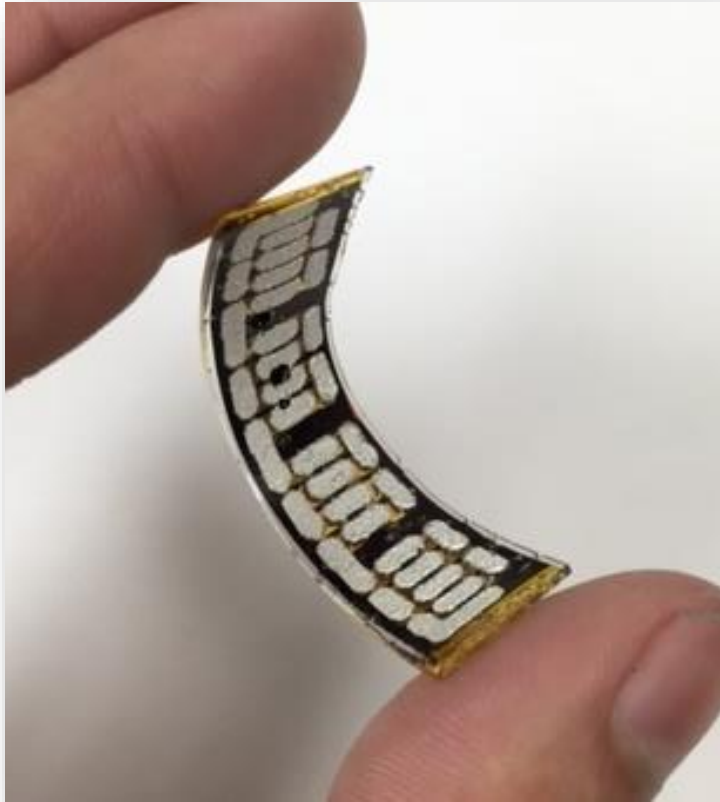
Thermal Energy Harvesting



Courtesy of Prof. Mehmet Ozturk, NC State



Advanced materials and processing

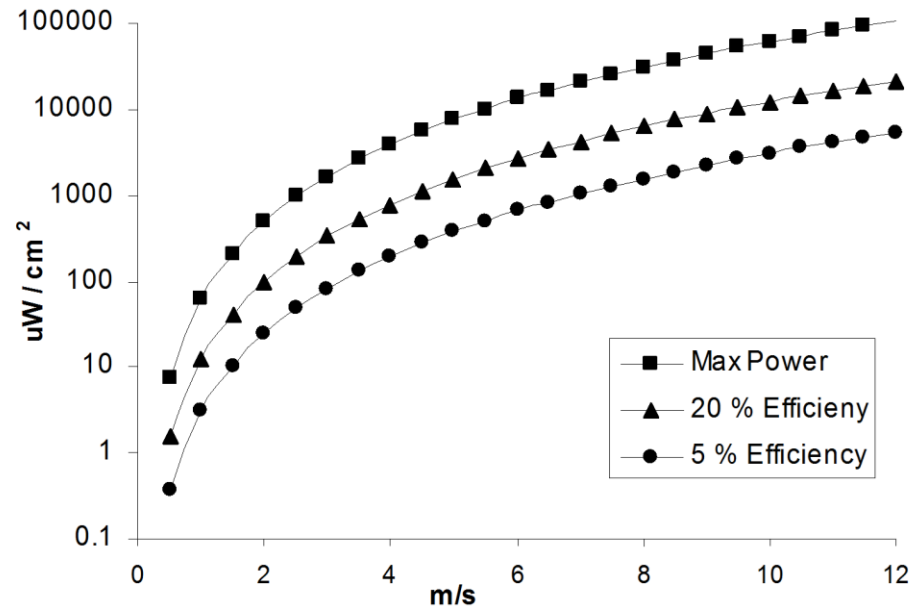


Courtesy of Prof.
Mehmet Ozturk,
NC State

Air / Fluid Flow

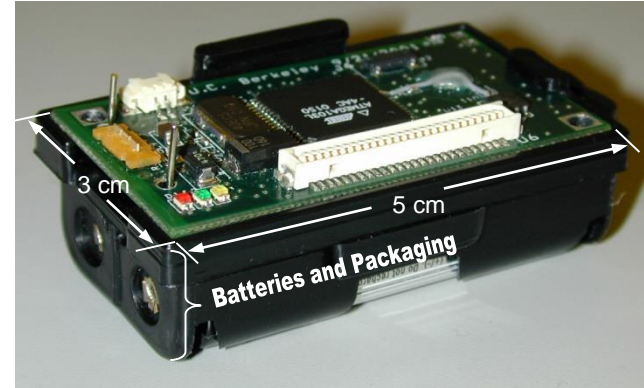
- Potential Power Given By:

$$P = \frac{1}{2} \rho A v^3$$



- Maximum Efficiency of Large Wind Turbines Generally About 40% (Limit Is 59%)
- Average Efficiency Generally About 20%
- Optimized for Wind Velocities of About 8 m/s

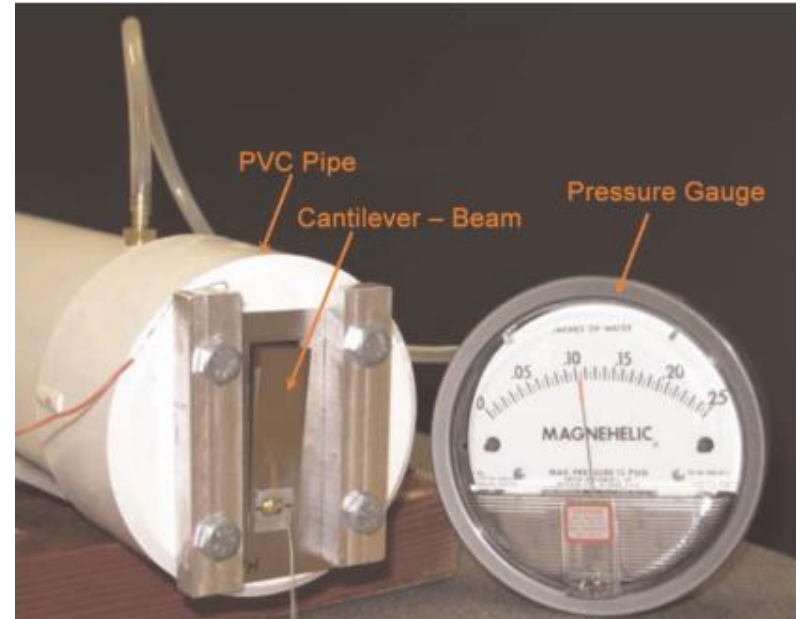
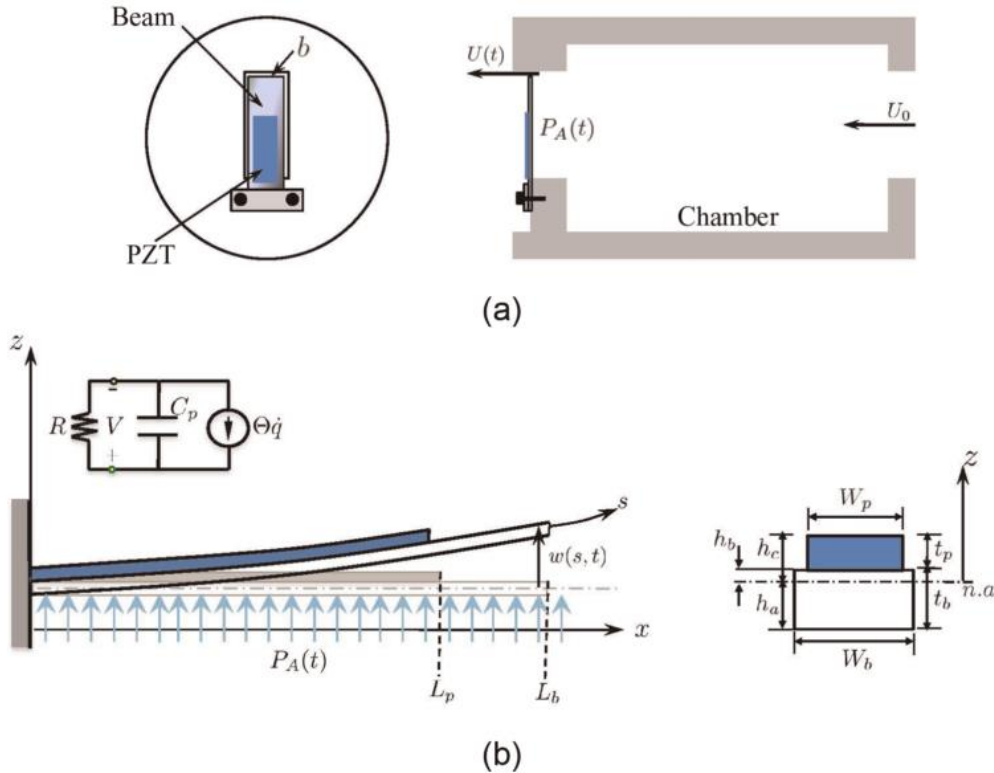
Air / Fluid Flow



Wind Velocity (m/s)	Power ($\mu\text{W}/\text{cm}^2$)	Efficiency
2.5	100	5 %
4	215	5.5 %
5	350	11 %

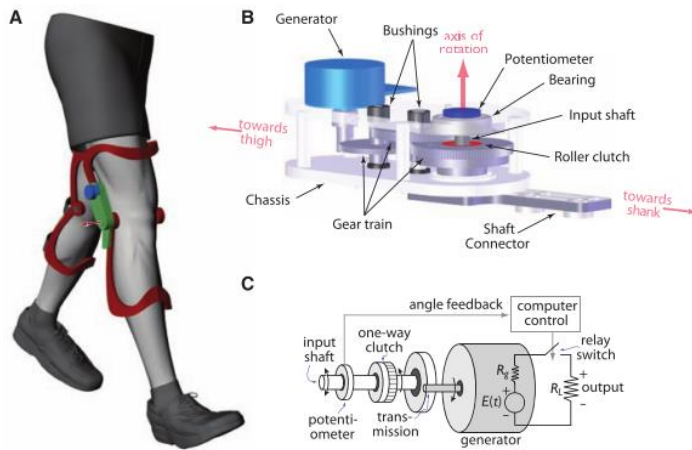
Experimental power output values reported by Federspiel and Chen, 2003

Aeroelastic Energy Harvesters



Bilbao, Liu, and Daqaq, JIMSS 2012

User Input

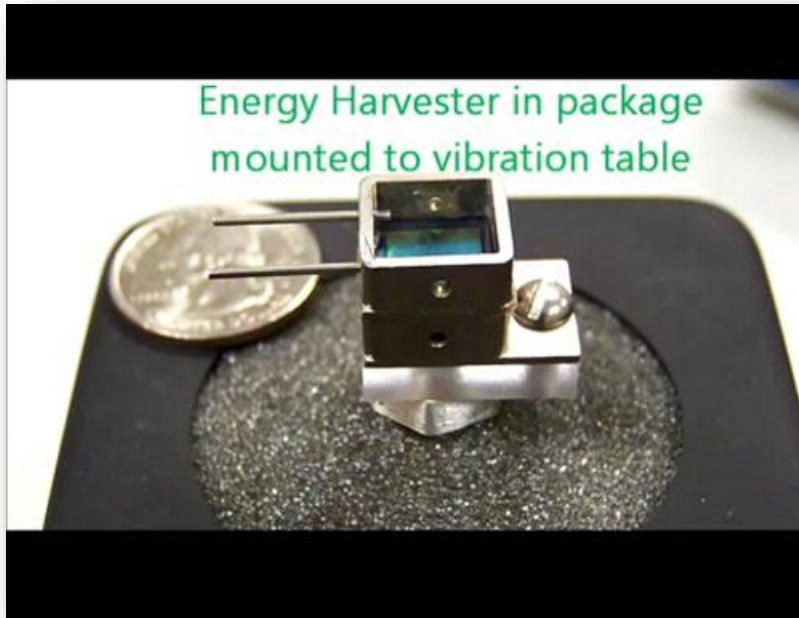


Donelan et. al. Science, 2008



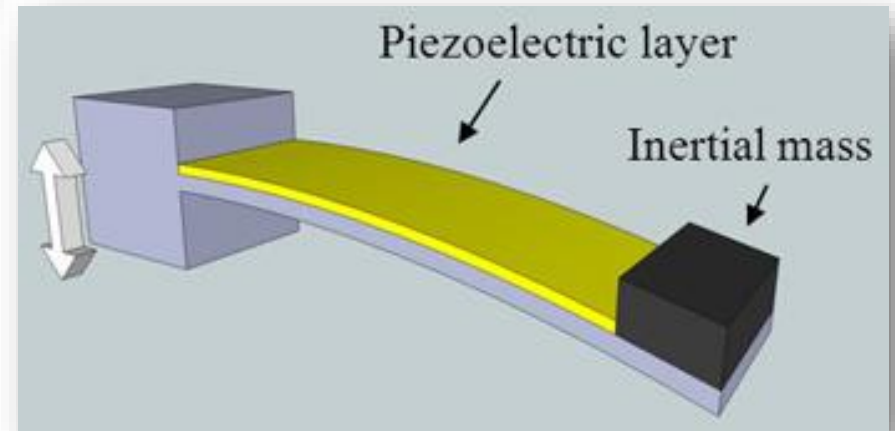
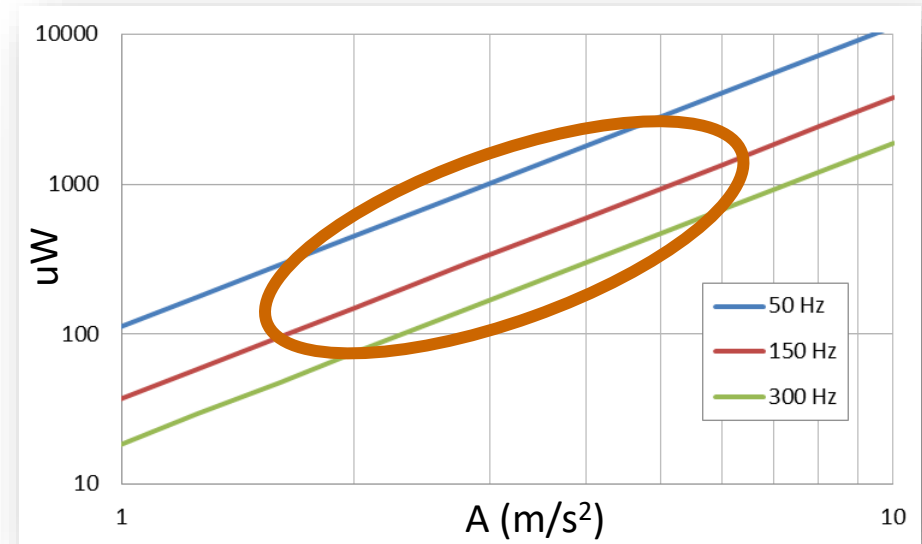
EnOcean, www.enocean.com

Vibrations



Microgen Systems Inc.

Courtesy of Robert Andosca

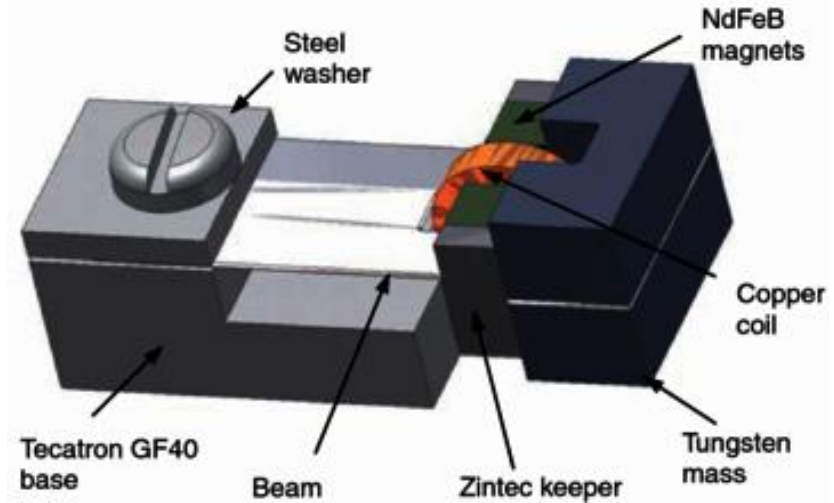


VIBRATION / MOTION BASED ENERGY HARVESTING

Perpetuum - Electromagnetic

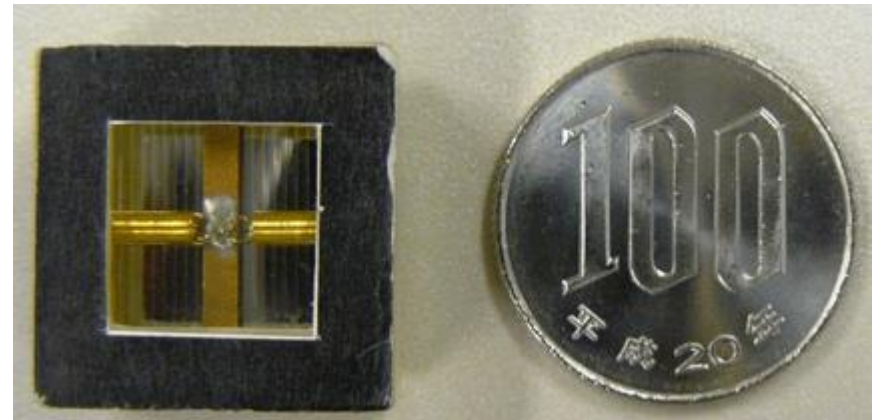
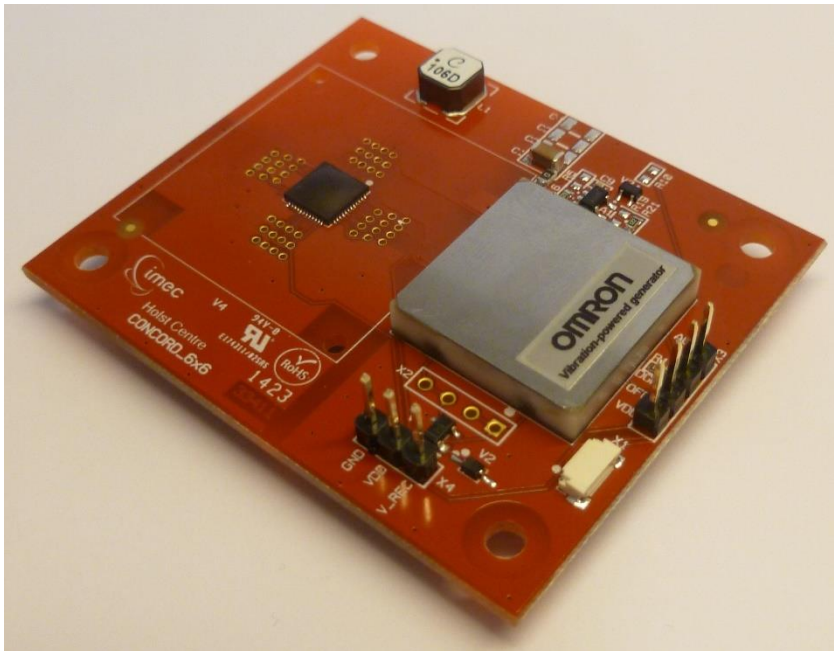


Perpetuum

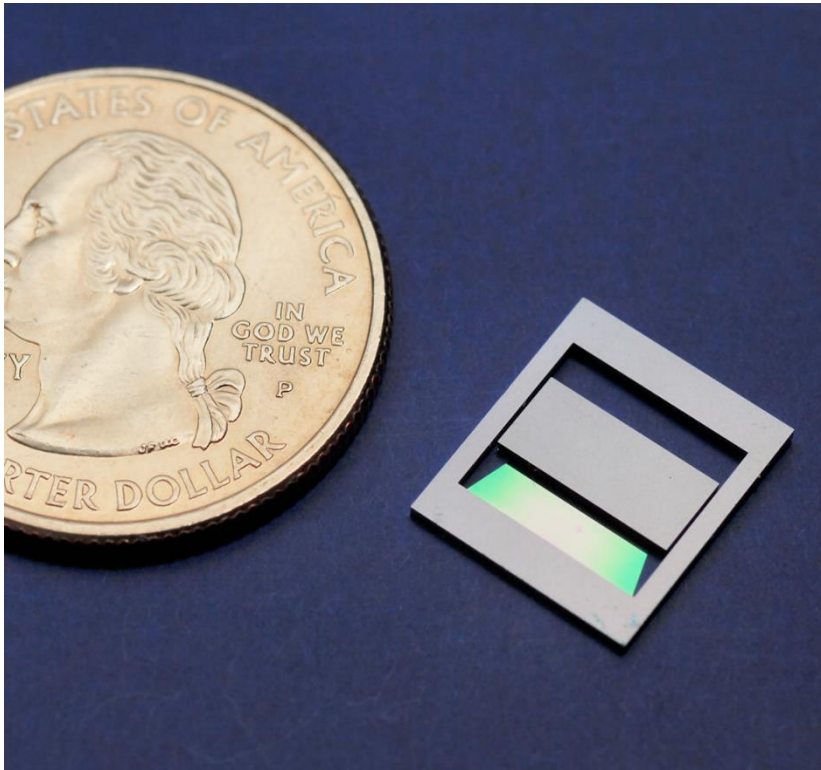


Beeby et. al, JMM, 2007

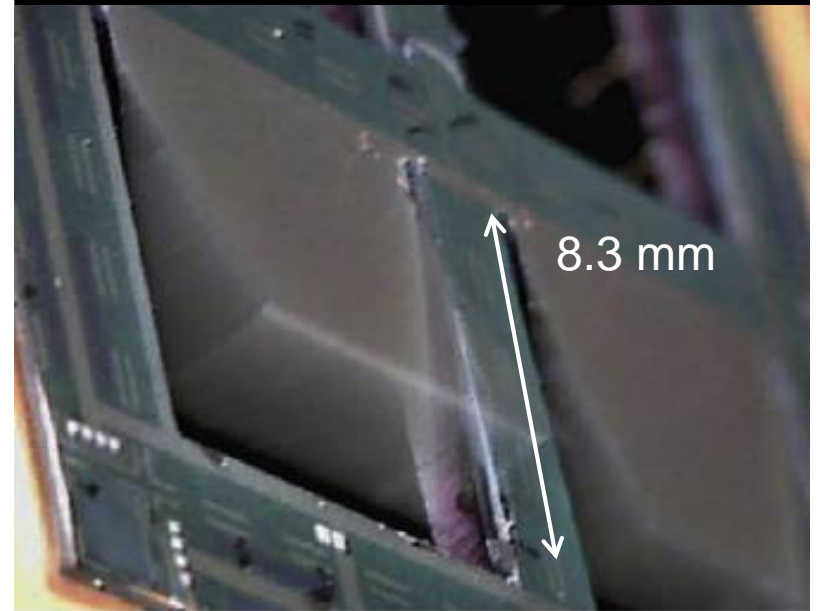
Omron - Electrostatic



MicroGen Systems - Piezoelectric

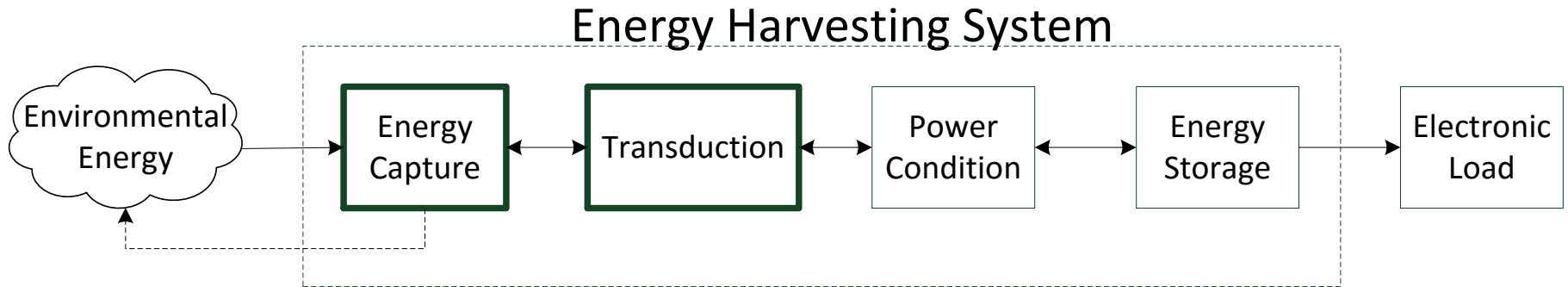


Andosca et al., Sensors and Actuators A, 178 (2012) 76



<https://www.microgensystems.com/>

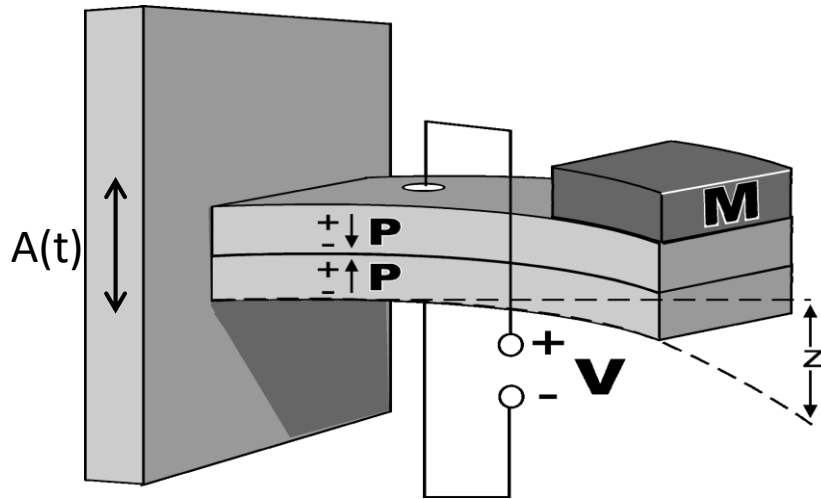
Energy Harvesting System Overview



“Micro Energy Harvesting”, Briand, et. al. 2015

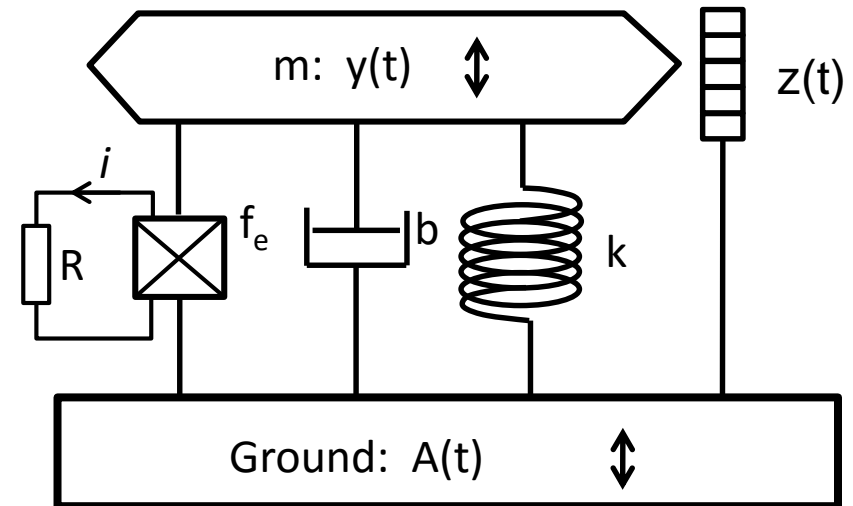
- An energy harvesting system consists of an energy capture mechanism (e.g. a cantilever beam, wind turbine), an electromechanical transducer (e.g. piezoelectric material), power conditioning circuitry, and usually temporary energy storage all of which delivers electrical power to a some electronic load.
- Each subsystem influences the behavior of the subsystem both immediately upstream and downstream in the overall system. The energy capture mechanism even affects the environment in which it operates, although this effect may be small.
- **This lecture primarily addresses the “Energy Capture” block**

Vibration Energy Harvesters (VEHs)



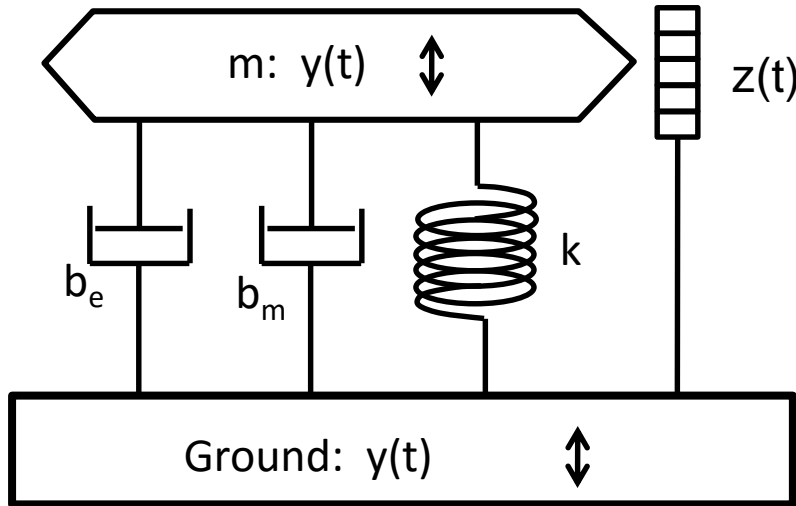
Roundy & Wright, SMS, 2004

- Piezoelectric bimorph beam VEH
- Base (or wall) is driven by vibration with acceleration $A(t)$
- There is a proof mass at the end of the beam, which flexes causing a 3-1 mode piezoelectric transducer



- Lumped parameter model of the piezoelectric beam VEH
- Mass is the equivalent mass (actual mass and inertial effects of beams)
- Piezo element creates an electrically induced force (f_e) on the mechanical oscillator as well as generating current through a load circuit

VDRG Vibration Energy Harvester Model



VDRG = Velocity Damped Resonant Generator

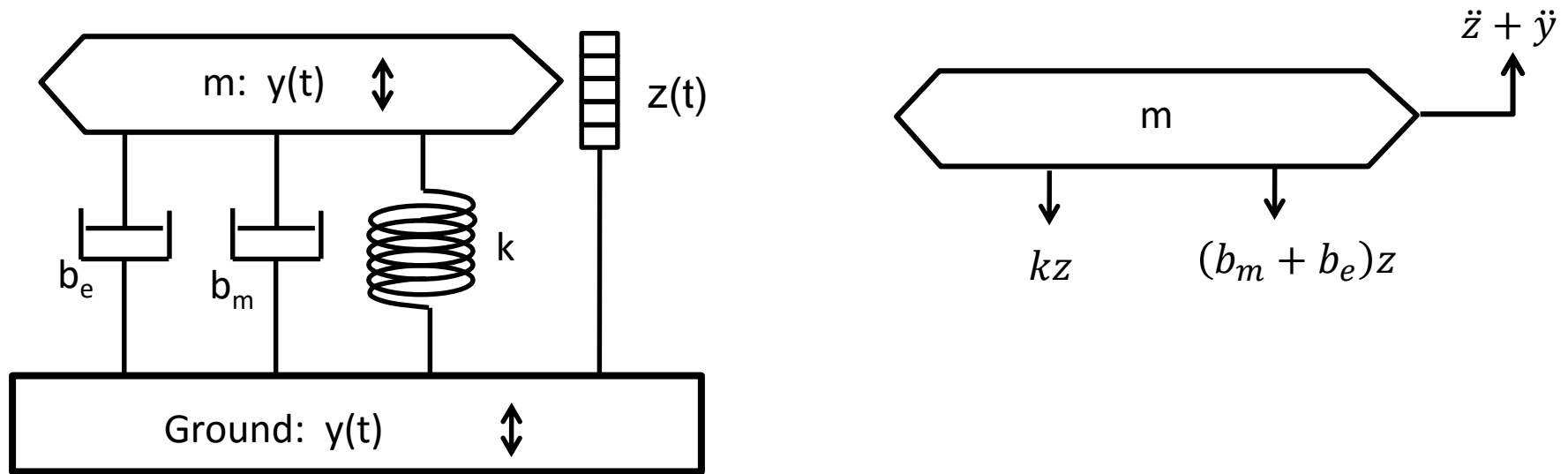
Mitcheson et. al., JMEMS 2004

Halvorsen et. al. J. Phys. Conf. Series, 2013

Heit & Roundy, En. Harv. and Sys., 2015

- b_e is the electrically induced damping coefficient
- Power dissipated through b_e is the power extracted by the load circuit
- This model has been shown to produce the upper bound on extractable power from an input dominated by a single, stable frequency

VDRG Vibration Energy Harvester Model



z is the relative displacement, so total acceleration experienced by proof mass is $\ddot{z} + \ddot{y}$

Governing equation:

$$m\ddot{z} + (b_m + b_e)\dot{z} + kz = -m\ddot{y}$$

VDRG Vibration Energy Harvester Model

Make the following substitutions to generalize the equation:

$$\omega_n^2 = \frac{k}{m} \quad \frac{b}{m} = 2\zeta\omega_n$$

where ω_n is the natural frequency and ζ is the dimensionless damping ratio

$$\ddot{z} + 2(\zeta_m + \zeta_e)\omega_n\dot{z} + \omega_n^2 z = -\ddot{y}$$

This is the standard forced oscillator equation, which is characteristic of a damped mass-spring system or an LCR circuit.

Solving for z in the frequency domain yields:

$$Z(j\omega) = \frac{\omega^2 Y}{\omega_n^2 - \omega^2 + j2\zeta\omega\omega_n}$$

where $\zeta = \zeta_m + \zeta_e$ is the total damping ratio.

VDRG Vibration Energy Harvester Model

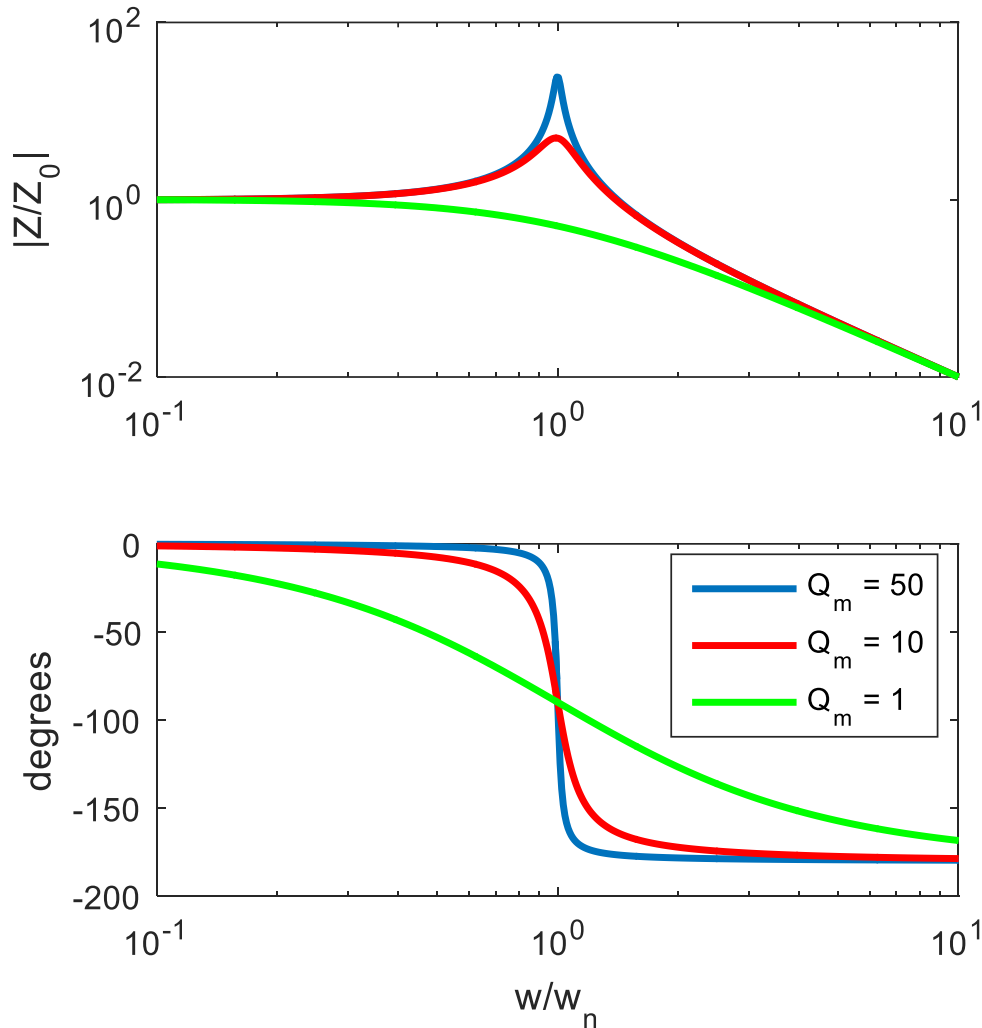
Let $r = \frac{\omega}{\omega_n}$ be the ratio of the driving frequency to the natural frequency, and:

$$Z(j\omega) = \frac{r^2 Y}{(1 - r^2) + j2\zeta r}$$

The magnitude of the response is:

$$Z_0 = |Z(j\omega)| = \frac{r^2 Y}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}$$

VDRG Vibration Energy Harvester Model



Note, the quality factor (or Q) is the amplification ratio at resonance, and is mathematically defined as $Q = \frac{1}{2\zeta}$, so the “mechanical Q” or $Q_m = 1/(2\zeta_m)$

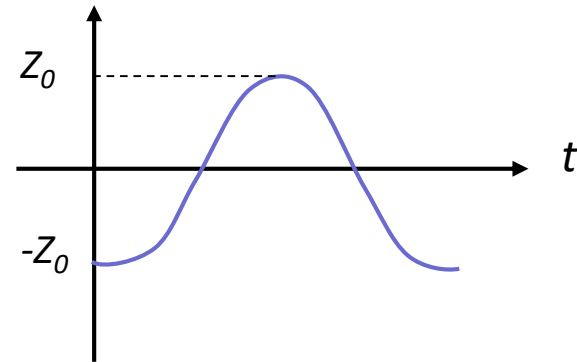
VDRG Vibration Energy Harvester Model

We want to determine how much power gets dissipated through electrical damper b_e .

At any instant, the electrically induced force on the proof mass is: $F(t) = b_e \dot{z}(t)$

To get the average RMS power, we will integrate $F(t)$ over one full displacement cycle to get the energy dissipated per cycle, E_{cyc} .

$$E_{cyc} = 2 \int_{-Z_0}^{Z_0} b_e \dot{z} dz = 2 \int_0^{\frac{\pi}{\omega}} b_e \dot{z}^2 dt$$



Then divide by the period, T , to get RMS power.

VDRG Vibration Energy Harvester Model

$$z(t) = Z_0 \sin(\omega t)$$

$$\dot{z}(t) = \omega Z_0 \cos(\omega t)$$

$$E_{cyc} = 2b_e \omega^2 Z_0^2 \int_0^{\frac{\pi}{\omega}} \cos^2(\omega t) dt$$

$$E_{cyc} = 2b_e \omega^2 Z_0^2 \frac{2\pi}{4\omega}$$

$$P_{rms} = \frac{E_{cyc}}{T} \quad \text{where } T = \frac{2\pi}{\omega}$$

$$P_{rms} = \frac{1}{2} b_e \omega^2 Z_0^2$$

VDRG Vibration Energy Harvester Model

Incorporate expression for Z_0

$$Z_0 = |Z(j\omega)| = \frac{r^2 Y}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$

$$P_{rms} = \frac{1}{2} b_e \frac{r^4 Y^2}{(1-r^2)^2 + (2\zeta r)^2}$$

Substituting $b_e = 2m\zeta_e\omega_n$

$$P_{rms} = \frac{m\zeta_e\omega^3 r^3 Y^2}{(1-r^2)^2 + (2\zeta r)^2}$$

More often written as a function of input vibration acceleration $A = Y\omega^2$:

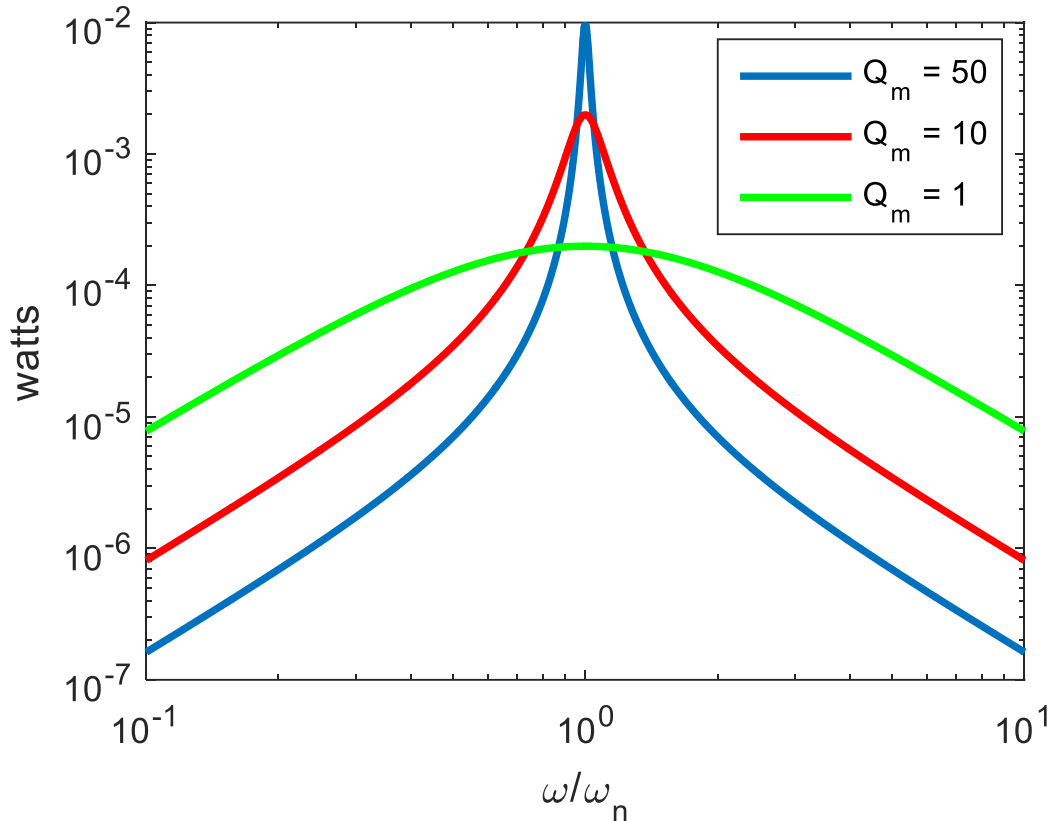
$$P_{rms} = \frac{m\zeta_e r^3 A^2}{\omega[(1-r^2)^2 + (2\zeta r)^2]}$$

Things to note from VDRG model

$$P_{rms} = \frac{m\zeta_e r^3 A^2}{\omega[(1-r^2)^2 + (2\zeta r)^2]}$$

- Maximum power occurs at resonance ($r = 1$)
- Power is proportional to proof mass
- Power is proportional to $\frac{A^2}{\omega}$ where ω is the driving frequency
- There is an optimal level of electrically induced damping ($\zeta_e = \zeta_m$)
 - Power is maximized by minimizing mechanical damping, and then matching electrical damping to the mechanical damping

Power vs. Frequency for Multiple Q's



$$A = 1 \text{ m/s}^2$$

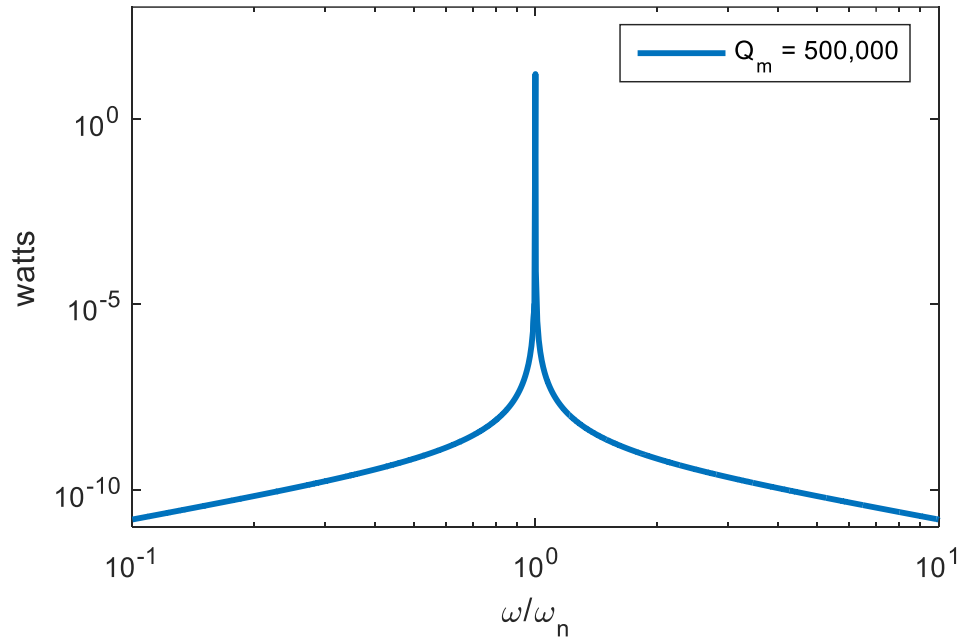
$$m = 1 \text{ kg}$$

$$\omega_n = 2\pi 100 \text{ rad/s}$$

Power at resonance is highly dependent on Q .

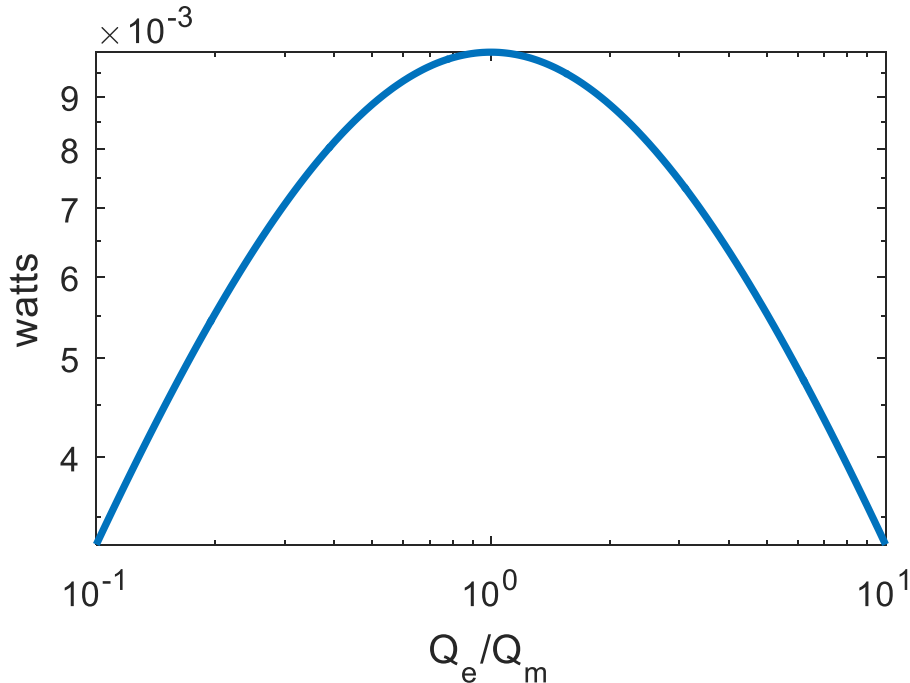
At high Q , where power is good, half-power bandwidth is extremely narrow, which is one of the big problems with vibration energy harvesters.

How High Can the Output Power Get?



- Technically, power output is unbounded
- Rests on the assumption that the vibration source will exert whatever force is necessary to maintain its displacement
- Rests on the assumption that proof mass displacement is also unbounded

Power vs. Electrical Damping



$$P_{rms}(r = 1) = \frac{m\zeta_e A^2}{4\omega\zeta^2}$$

Assume resonance

$$Q_m = 50$$

Power is somewhat sensitive to electromechanical coupling which relates to Q_e .

In practice most, though not all, systems are under-coupled and so on the left side of this graph.

Example 1: Belt worn harvester

- $m = 6.1 \text{ oz} = 173 \text{ grams}$ (iPhone 6)
- Walking motion is about 1 Hz
- $A = 0.5 \text{ g} = 5 \text{ m/s}^2$
- Conservatively assume $Q_m = 20$, $Q_e = 20$
- Assume resonance

Example 1: Belt worn harvester

- $m = 6.1 \text{ oz} = 173 \text{ grams}$ (iPhone 6)
- Walking motion is about 1 Hz
- $A = 0.5 \text{ g} = 5 \text{ m/s}^2$
- Conservatively assume $Q_m = 20, Q_e = 20$
- Assume resonance

$$P_{rms}(r = 1) = \frac{m\zeta_e A^2}{4\omega\zeta^2} = \frac{mQ^2 A^2}{2\omega Q_e}$$

- $P = 1.7 \text{ watts} \text{ !!!!}$

Example 1: Belt worn harvester

- But, motion is unconstrained, what is Z_0 ?

$$Z_0 = |Z(j\omega)| = \frac{r^2 Y}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}$$

$$Y = \frac{A}{\omega^2} = 12.7 \text{ cm}$$

$$Z_0(r = 1) = \frac{A}{2\omega^2\zeta} = \frac{Y}{Q}$$

- $Z_0 = 1.27$ meters !!!

Example 1: Belt worn harvester

- So, there is a lot of power available, but unfortunately it requires very large displacements.
- This, along with bandwidth, is the most challenging issue with body worn inertial harvesters
- See for more discussion of the effect of displacement limits:
 - Mitcheson et. al. 2004 “Architectures for Vibration-Driven Micropower Generators”, *Journal of Microelectromechanical Systems*, **3**, pp 1-12, 2004.
 - Mitcheson et. al. 2008 “Energy Harvesting from Human and Machine Motion for Wireless Electronic Devices”, *Proceedings of the IEEE*, vol. 96 **9**, pp1457-1486.

Example 2

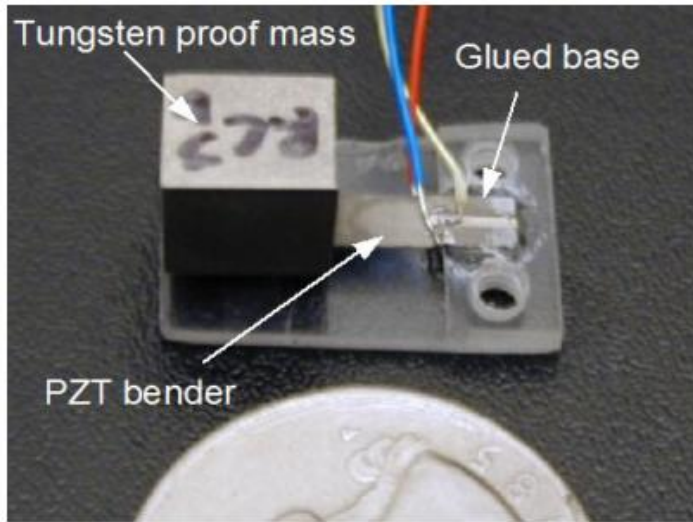


Figure 14. The optimized piezoelectric generator with a 1.5 cm length constraint (design 1).

Roundy and Wright, SMS 2004

$$m = 7.45 \text{ grams}$$

$$\omega_n = 2 \cdot \pi \cdot 120 \text{ rad/s}$$

$$A = 2.5 \text{ m/s}^2$$

$$\zeta_m = 0.015 \text{ (} Q_m = 33\text{)}$$

According to VDRG,
max power = $255 \mu\text{W}$

Measured power = $210 \mu\text{W}$

$$Z_0 = 73 \mu\text{m}$$

$$\text{“effectiveness”} = P_{\text{meas}}/P_{\text{VDRG}} = 82\%$$

See Mitcheson et. al. 2008, Proceedings of the IEEE

Figures of Merit

$$P_{rms}(r = 1) = \frac{m\zeta_e A^2}{4\omega\zeta^2}$$

- Different types of power densities are often reported
 - $\text{W/cm}^3/\text{g}^2$ or $\text{W/cm}^3/\text{g}^2/\text{Hz}$
 - While these are reasonable metrics for comparison, they only tell part of the story.
- My preference for FOM is that defined by Mitcheson:
 - Harvester effectiveness: $E_H = \frac{\text{Useful Power Out}}{P_{VDRG}}$
 - This is really a measure of how well a particular design performs relative to its ideal performance

Graph of Reported Effectiveness (2008)

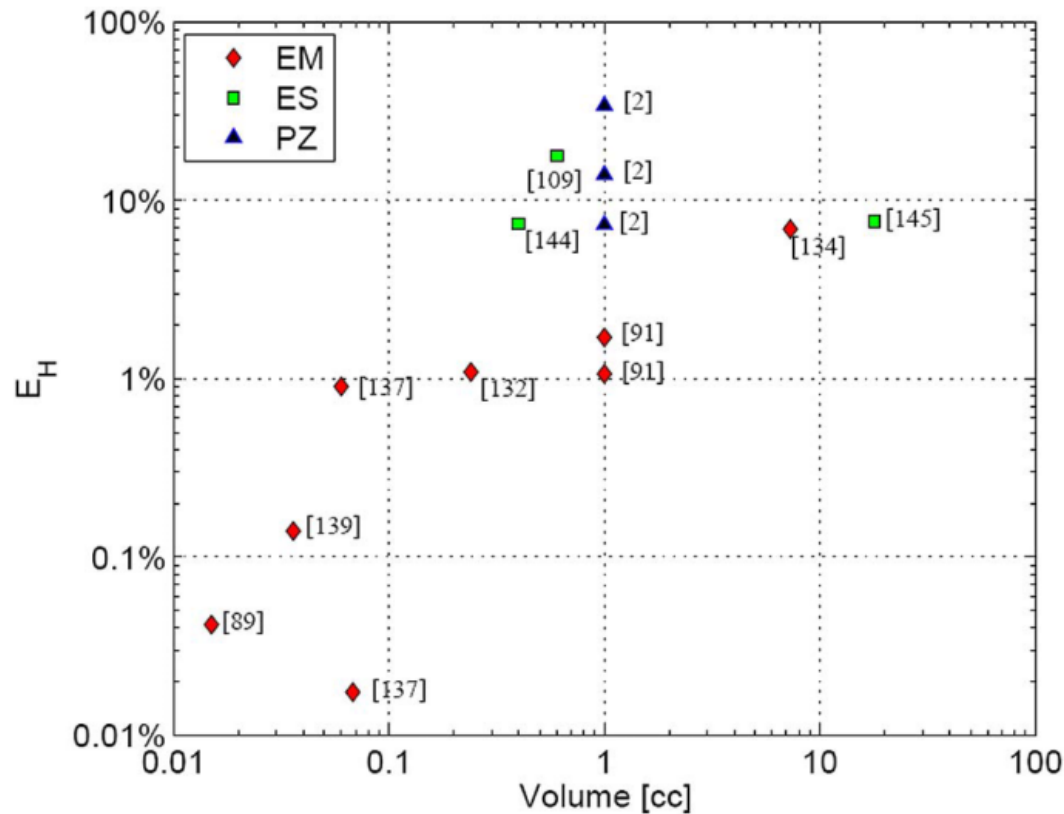


Fig. 27. Harvester effectiveness of reported devices versus device volume.

Mitcheson et. al. 2008 “Energy Harvesting from Human and Machine Motion for Wireless Electronic Devices”, *Proceedings of the IEEE*, vol. 96 9, pp1457-1486.

Next Up

- Here, we just lumped all electromechanical transduction into a single damping parameter
- Next lecture will go over specific transduction mechanisms